# Femlab 3.0: Experiences in Determining RTD 

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## Introduction

The use of commercially available Computational Fluid Dynamics (CFD) packages to study the residence time distribution (RTD) in wastewater treatment plants was recently discussed (Madeira, et. al , 2004). In that study the Fluent package (ver 6.0) ( http://www.fluent.com) was used to simulate a 2-D reservoir with dimensions L (length) and H (height) in laminar flow and isothermal conditions (see Figure 1).


Figure 1 Sketch of Reservoir Geometry
(From Madeira et. al., 2004 with permission) ${ }^{1}$

Both the inlet and the outlet boundaries of the reservoir have a height of 0.01 m with distances from the bottom of the reservoir of 0 an 0.02 m , respectively. Both L and $\mathrm{H}=0.1 \mathrm{~m}$.

A fully developed parabolic velocity profile is imposed on the inlet boundary

$$
\begin{equation*}
\mathrm{u}_{\mathrm{x}}=\operatorname{Umax}\left[1-\{(\mathrm{y}-\mathrm{H} / 20) / \mathrm{H} / 20\}^{2}\right] \tag{1}
\end{equation*}
$$

where

$$
\operatorname{Umax}=1.5 \text { Umean }
$$

There is no slip at the walls $\left(u_{x}=u_{y}=0\right)$.
A step input of a component with properties similar to water ( $\eta=0.001 \mathrm{~kg} /(\mathrm{m}-\mathrm{s}), \rho=1000$ $\mathrm{kg} / \mathrm{m}^{3}$, with $\left.\mathrm{D}=5 \mathrm{E}-10 \mathrm{~m}^{2} / \mathrm{s}\right)$ and having constant concentration ( $1000 \mathrm{~kg} / \mathrm{m}^{3}$ ) across the inlet boundary is introduced at time zero.

## Results of the Study with Fluent

Figures 2 and 3 indicate the results of the studies by Madeira et. al., 2004 at various Reynolds numbers with $\mathrm{L} / \mathrm{H}=1$. They present a table showing the influence of Reynolds number on the mean residence time.


Figure 2. Steady state contours of the stream function for the reservoir with $\mathrm{L} / \mathrm{H}=1$ as a function of Reynolds number
(From Madeira et. al., 2004 with permission) ${ }^{1}$


Figure 3 Effect of Reynolds number on the residence time distribution $(\mathrm{L} / \mathrm{H}=1)$
(From Madeira et. al. 2004 with permission) ${ }^{1}$
It is the purpose of this study to see how the same problem is solved using Femlab 3.0 (http://www.comsol.com). The study is limited to $\mathrm{Re}=10$ and $\mathrm{L} / \mathrm{H}=1$.

## Setting up Femlab

The general procedure followed using Femlab is:

1. Solve the ns (Navier-Stokes) equation at steady state
2. Solve the cd (Convection-Diffusion) equation starting at time zero to time 500 sec in time increments of 2 sec holding constant the steady state solution of the ns equation.
3. Record the integrated exit boundary concentration at the specified times on a spreadsheet for analysis of the RTD and mean residence time.

In Femlab Enter:
New, 2D
Chemical Engineering Module
Momentum Balance
Incompressible Navier-Stokes
Steady State Analysis (highlighted)
Click Multiphysics then Add
Mass Balance
Convection and Diffusion
Transient Analysis (highlighted)
Click Add
Click OK

## Setting Up the Geometry

In Femlab enter with respect to Figure $1(\mathrm{H}=0.1, \mathrm{~L}=0.1)$ :
Draw
Specify Objects
Line
Enter the line coordinates for each boundary of the geometry Click on Zoom Extents (on the menu) after each line is entered.
Coerce to Solid
The inlet flow boundary is assigned as boundary 1 , the outlet boundary is assigned as boundary 6 , the top left boundary as 3 , the top boundary as 4 , the top right boundary as 7 , the bottom right boundary as 5 and the bottom boundary as 2 .

## Setting up the Constants

In Femlab enter:

## Options

Constants


Figure 4. Femlab Constants window

Note (Figure 4) that eta $=\eta$, den $=\rho$, Difus is D (the diffusivity) and Co is the concentration $\left(\mathrm{kg} / \mathrm{m}^{3}\right)$ of the component at the inlet. Tau $(\tau)$ is the space- time and. $\mathrm{U}_{\max }$ is calculated from the Reynolds number as shown.

## Solving the Navier-Stokes Equations

In Femlab enter (Multiphysics -ns):
Options
Expressions
Boundary Expressions Boundary Expression 1

Name Expression uvel Umax * [1-((y-HH)/HH)^2]

Physics
Subdomain Settings 1
Physics Tab
$\rho=$ den Density
$\eta=$ eta Dynamic Viscosity

Boundary Settings
Boundary Setting 1
Boundary Condition Inflow/Outflow Velocity
$u_{0}=$ uvel $\quad x$ velocity
$v_{0}=0 \quad y$ velocity
Boundary Setting 2
Boundary Condition No Slip
Boundary Setting 3
Boundary Condition No Slip
Boundary Setting 4
Boundary Condition No Slip
Boundary Setting 5
Boundary Condition No Slip
Boundary Setting 6
Boundary Condition OutFlow/ Pressure
Boundary Setting 7
Boundary Condition No Slip
Mesh
Initial Mesh
Refine Mesh
Refine Mesh
(A finer mesh size was tried but memory was found to be inadequate ( 512 MB ) ).

```
Solve
    Solver Parameters
    Stationary Nonlinear
    Solver Manager
        Initial Value Tab
            Initial Value
            Check Initial Value Expressions
            Fixed Solution/Linearization Initialization Point
            Check Initial Value
    Solve for Tab
            Geom 2D
            Incompressible Navier-Stokes (highlight)
Output Tab
            Geom 2D
            Incompressible Navier-Stokes (highlight)
Solve Problem
Postprocessing
    Plot Parameters
        General Tab
            Check Streamline
        Streamline Tab
            Check Number of Start Points (60)
        Click OK
                            Reynolds Number = 10
```



Figure 5 Streamlines of the solution at $\mathrm{Re}=10$
This output compares well to the Fluent results of Figure 2 at $\mathrm{Re}=10$.

## Solving the Convection-Diffusion Equation

The solution to the time dependent convection-diffusion equation (cd) proved to require considerable trial and error since the problem is so convection dominant (Comsol_a, 2004). Femlab provides a number of artificial stabilization methods (Comsol_b, 2004) which modifies the original problem. Some of the methods, however, do not provide what were judged to be reasonable solutions.

When no stabilization is used, Figure 6 is the concentration profile that occurs after 222 sec . and Figure 7 is the results after 398 sec . A negative value for the integrated concentration at the exit at 222 sec is returned and the profile at 398 sec does not seem reasonable.


Figure 6. Exit concentration profile at 222 sec with no stabilization


Figure 7 Exit Concentration Profile at 398 sec - No stabilization

Both streamline diffusion and crosswind diffusion stabilization methods were tried but rejected. Isotropic diffusion was most extensively tested and was finally accepted. The tuning parameter was slowly decreased from a value of 0.5 to 0.008 , the point at which the concentration profiles were still acceptable (Figures 8 and 9).


Figure 8 Exit concentration profile at 222 sec with isotropic stabilization $=0.008$


Figure 9 Exit Concentration Profile at 398 sec with isotropic stabilization $=0.008$

The procedure with isotropic stabilization to solve the cd equation (storing the steady state ns solution):

Enter Femlab:

Multiphysics
Convection and Diffusion (highlighted)
Physics
Boundary Settings
Boundary Selection 1 Concentration
Co = Co
Boundary Selection 2 Insulation/Symmetry
Boundary Selection 3 Insulation/Symmetry
Boundary Selection 4 Insulation/Symmetry
Boundary Selection 5 Insulation/Symmetry
Boundary Selection 6 Convective Flux Boundary Selection 7 Insulation Symmetry

Subdomain Settings
c Tab

| $D$ Isotropic | Difus | Diffusion Coeffiscient |
| :--- | :--- | :--- |
| $u$ | $u$ | $x$ velocity |
| $v$ | $v$ | $y$ velcity |

Artificial Diffusion isotropic diffusion $\delta_{\text {id }}=0.008$ Tuning Parameter

Init Tab
$c\left(t_{0}\right) \quad 0$ Concentration $c$
Solve
Solver Parameters
General Tab
Solver - Time Dependent
Time Stepping - Time 0:2:500
Time Stepping Tab
Time step taken by Solver - Strict
Some differences were noted in how the time steps were determined (fixed, free) but the differences were not judged to be significant.

Solver Manager

## Click Store Solution

Initial Value Tab
Initial Value
Click Initial value expression evaluated using stored solution
Fixed solution/Linearization point
Click Stored Solution
Solve for Tab
Geom(2D)
Convection and Diffusion (highlight)
Output Tab
Geom 2D
Incompressible Navier-Stokes (highlight)
Convection and Diffusion (highlight)

Solve
Click Solve Problem
Time of Solution : 677.9 sec Degrees of Freedom 100, 088

## Postprocessing

The solution to the steady state ns equation and the transient cd equation can be viewed in a number of ways in the postprocessing step. Figure 10 shows the concentration of the tracer after 222 sec at which point the tracer is just reaching the exit. This is consistent with a similar plot presented by Maderia et. al., 2004.

Enter Femlab:
Postprocessing
General Tab
Check Surface


Figure 10 Surface tracer concentration for the reservoir after 222 sec

$$
(\operatorname{Re}=10, L / H=1)
$$

The cross sectional plots of Figures 6, 7, 8 and 9 were obtained at $\mathrm{x}=0.0995 \mathrm{~m}$ ( $\mathrm{y}=0.02$ to 0.03 ) rather than at $\mathrm{x}=0.1$ since a bug exists in Version 3.0 (Comsol_c, 2004).

Enter Femlab:

Postprocessing
Cross Sectional Plots
General (time selected)
Line Extension (cross section line data)

## Spreadsheet Calculations

After the calculation for the cd equation is complete, postprocessing is accessed and boundary integration is requested:

Enter Femlab:

Postprocessing
Boundary Integration
Boundary Selection 6
Time (selected)
The integrated concentration at boundary 6 (the exit) is returned on the screen at the requested time and is entered manually into the spreadsheet (Appendix A) column 'Cout/Integrated'. The Danckwert's F curve values ( $\mathrm{C}_{\mathrm{out}} /(0.01 * \mathrm{Co})$ are calculated in the next column where the 0.01 term is the height of the exit.

The resident-time distribution function

$$
\begin{equation*}
\mathrm{E}(\mathrm{t})=\mathrm{d}(\mathrm{~F}(\mathrm{t})) / \mathrm{dt} \tag{2}
\end{equation*}
$$

is calculated using an average of the forward and backward derivative of the $\mathrm{F}(\mathrm{t})$ values at a particular time (Chapra et al, 1988). Normalizing

$$
\begin{equation*}
\mathrm{E}(\theta)=\mathrm{E}(\mathrm{t}) / \tau \tag{3}
\end{equation*}
$$

where the space-time is

$$
\begin{equation*}
\tau=\text { Area } / \text { Flow Rate }=(0.1) *(0.1) /(.001 * .01)=1000(1 / \mathrm{s}) \tag{4}
\end{equation*}
$$

Figure 11 is a plot of the residence time distribution ( $\mathrm{E}(\theta)$ vs $\theta$ ) out to $\theta=0.398$. The plot closely matches the results of Madeira et. al., 2004 except that the tail becomes unstable (compare Figure 3).


Figure 11 Residence time distribution for $\operatorname{Re}=10, \mathrm{~L} / \mathrm{H}=1$

## Determination of Reduced Mean Residence Time

The mean reduced residence time $\bar{\theta}_{\mathrm{r}}$ is given (Himmelblau et. al. 1968) by

$$
\begin{equation*}
\bar{\theta}_{\mathrm{r}}=\bar{t}_{\mathrm{r} / \tau}=\int_{0}^{\infty} \theta E(\theta) \mathrm{d} \theta \tag{5}
\end{equation*}
$$

where

$$
\begin{equation*}
\bar{t}_{\mathrm{r}}=\int_{0}^{\infty} \mathrm{tE}(\mathrm{t}) \mathrm{dt} \tag{6}
\end{equation*}
$$

Since the tail of the RTD was unstable, determination of the area under the curve was broken into two parts: from $\theta=0.208$ to $\theta=0.398$ and from $\theta=0.398$ to $\infty$. The area of the first part of the curve was determined by using the trapezoidal rule (Appendix A). The trapezoidal rule was also used for the $\theta^{*} \mathrm{E}(\theta)$ curve.

A plot of $\ln (\mathrm{E}(\theta))$ vs $\theta$ is shown in Figure 12.


Figure 12. Plot of $\ln (\mathrm{E}(\theta))$ vs $\theta$

The plot indicates that a curve of the form

$$
\begin{equation*}
\mathrm{E}(\theta)=\mathrm{A} \exp (-\mathrm{B} * \theta) \tag{7}
\end{equation*}
$$

could approximate the tail of the curve from $\theta$ from 0.398 to $\infty$.

The areas of both he $\mathrm{E}(\theta)$ and $\theta * \mathrm{E}(\theta)$ curve can then be calculated analytically from the following:

Noting that

$$
\begin{equation*}
\int_{0}^{\infty} E(\theta) d \theta=1 \tag{8}
\end{equation*}
$$

the area under the curve from $\theta=0.398$ to $\infty$ must be 1- $0.6590=0.3410$ where the 0.6590 is the area from $\theta=0.208$ to 0.398

From Equation (7)

$$
\begin{equation*}
\int_{0.398}^{\infty} \mathrm{A} \exp \left(-\mathrm{B}^{*} \theta\right) \mathrm{d} \theta=(\mathrm{A} / \mathrm{B}) \exp \left(-\mathrm{B}^{*} 0.398\right)=0.3410 \tag{9}
\end{equation*}
$$

From the spreadsheet at $\theta=0.398$

$$
\begin{equation*}
\mathrm{E}(\theta)=1.1875=\mathrm{A} \exp (-\mathrm{B} * 0.398) \tag{10}
\end{equation*}
$$

Using Equations (9) and (10)

$$
\begin{equation*}
B=1.1875 / 0.3410=3.48240 \tag{11}
\end{equation*}
$$

and from Equation (7)

$$
\begin{equation*}
\mathrm{A}=1.1875 / \exp (-.3 .48240 * 0.398)=4.748579 \tag{12}
\end{equation*}
$$



Figure 13 Plot of $\ln \mathrm{E}(\theta)$ vs $\theta$ with approximate tail added

With the values of $A$ and $B$ available the tail of the $\theta * E(\theta)$ curve can be calculated analytically using equation (7):

$$
\begin{equation*}
\int_{.398}^{\infty} \theta E(\theta) \mathrm{d} \theta=\left(\mathrm{A} / \mathrm{B}^{2}\right) *[(\exp (-0.398 * \mathrm{~B}) *(-\mathrm{B}(.398)-1)]=0.2337 \tag{13}
\end{equation*}
$$

When the tail value is added to the value of the integral calculated in the spreadsheet

$$
\begin{equation*}
\text { Total area under } \theta^{*} \mathrm{E}(\theta) \text { curve }=0.1798+0.2337=\mathbf{0 . 4 1 3 5} \tag{14}
\end{equation*}
$$

This compares very well with the value of the mean residence time given my Madeira et el, 2004 of $\mathbf{0 . 4 1 2}$.

## Conclusions

The solution presented in this study is reasonable but could no doubt be improved. The solution presented by Madeira et. al., 2004. (Figure 2) did not appear to require any special processing for the tail and subsequent determination of the mean residence time by integration up to $\theta=10$. Such a smooth tail could not be generated in this study using the application software.

Madeira et. al., 2004 did mention, however, that a special discretization scheme (Versteeg et. al, 1995) was used for the convective terms.

Madiera et. al., 2004 states "We must point out that to achieve a high level of accuracy, all the simulation results presented in his paper involved a detailed analysis of the numerical algorithms, the mesh employed, and the time step adopted (in transient simulations)". A similar statement can perhaps be made with regard to Femlab 3.0.

## Nomenclature

| A, B | Parameters in Eq 7 |
| :--- | :--- |
| C | Concentration $\left(\mathrm{kg} / \mathrm{m}^{3}\right)$ |
| d | Inlet boundary height $(\mathrm{m})$ |
| D | Diffusivity $\left(\mathrm{m}^{2} / \mathrm{s}\right.$ |
| $\mathrm{E}(\mathrm{t})$ | Resident-time distribution function |
| $\mathrm{E}(\theta)$ | Normalized RTD function, dimensionless |
| $\mathrm{F}(\mathrm{t})$ | Danckwerts' F curve, dimensionless |
| H | Height of reservoir, $(\mathrm{m})$ |
| HH | $=\mathrm{H} / 20$ |
| L | Length of reservoir, $(\mathrm{m})$ |
| Q | Flow rate $\left(\mathrm{m}^{2} / \mathrm{s}\right)$ |
| Re | Reynolds number, $\mathrm{dU}_{\text {mean }} \rho / \eta$ |
| t | Time (s) |
| $t_{\mathrm{r}}$ | mean residence time $(\mathrm{s})$ |
| Umax | fluid velocity at the center of the inlet boundary $(\mathrm{m} / \mathrm{s})$ |
| Umean | mean fluid velocity at the inlet boundary $(\mathrm{m} / \mathrm{s})$ |
| $\mathrm{u}_{\mathrm{x}}$ | x-velocity $(\mathrm{m} / \mathrm{s})$ |
| $\mathrm{u}_{\mathrm{y}}$ | y-velocity $(\mathrm{m} / \mathrm{s})$ |
| x | horizontal coordinate $(\mathrm{m})$ |
| y | vertical coordinate $(\mathrm{m})$ |

## Greek Symbols

| $\delta_{\text {id }}$ | isotropic diffusion tuning parameter |
| :--- | :--- |
| $\eta$ | dynamic viscosity $(\mathrm{kg} / \mathrm{m}-\mathrm{s})$ |
| $\theta$ | $=\mathrm{t} / \tau$, reduced time, dimensionless |
| $\bar{\theta}_{\mathrm{r}}$ | reduced mean residence time, dimensionless |
| $\rho$ | density $\left(\mathrm{kg} / \mathrm{m}^{3}\right)$ |
| $\tau$ | space-time $(\mathrm{s})$ |

## References

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## Appendix A - The Spreadsheet

Residence Time Calculations - CEE Spring 2004 p154

| $\tau$ | 1000 | $\delta_{\text {id }}$ | 0.008 |
| :--- | :--- | :--- | :--- |
| $\mathrm{C}_{0}$ | 1000 |  |  |


| Time | $\theta=\mathrm{t} / \tau$ | Cout <br> sec |  | $\mathrm{F}(\mathrm{t})=$ <br> Integrated | $\mathrm{E}(\mathrm{t})$ | $\mathrm{E}(\theta) /\left(0.01^{*} \mathrm{C}_{0}\right)$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\theta^{*} \mathrm{E}(\theta)$ | $\ln [\mathrm{E}(\theta)]$ |  |  |  |  |  |  |


| 206 | 0.206 | $1.078 \mathrm{E}-05$ | $1.078 \mathrm{E}-06$ |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 208 | 0.208 | $2.080 \mathrm{E}-05$ | 2.080E-06 | $1.700 \mathrm{E}-06$ | 0.00170 | 0.00035 | -6.3772 |
| 210 | 0.21 | 7.877E-05 | 7.877E-06 | $2.899 \mathrm{E}-06$ | 0.00290 | 0.00061 | -5.8435 |
| 212 | 0.212 | $1.368 \mathrm{E}-04$ | $1.368 \mathrm{E}-05$ | $1.840 \mathrm{E}-05$ | 0.01840 | 0.00390 | -3.9955 |
| 214 | 0.214 | 8.147E-04 | 8.147E-05 | $5.508 \mathrm{E}-05$ | 0.05508 | 0.01179 | -2.8989 |
| 216 | 0.216 | $2.340 \mathrm{E}-03$ | $2.340 \mathrm{E}-04$ | $1.210 \mathrm{E}-04$ | 0.12098 | 0.02613 | -2.1121 |
| 218 | 0.218 | $5.654 \mathrm{E}-03$ | 5.654E-04 | $3.826 \mathrm{E}-04$ | 0.38263 | 0.08341 | -0.9607 |
| 220 | 0.22 | $1.765 \mathrm{E}-02$ | $1.765 \mathrm{E}-03$ | $1.347 \mathrm{E}-03$ | 1.34713 | 0.29637 | 0.2980 |
| 222 | 0.222 | 5.954E-02 | $5.954 \mathrm{E}-03$ | $3.695 \mathrm{E}-03$ | 3.69452 | 0.82018 | 1.3068 |
| 224 | 0.224 | $1.654 \mathrm{E}-01$ | $1.654 \mathrm{E}-02$ | $7.641 \mathrm{E}-03$ | 7.64070 | 1.71152 | 2.0335 |
| 226 | 0.226 | 3.652E-01 | 3.652E-02 | $1.224 \mathrm{E}-02$ | 12.23534 | 2.76519 | 2.5043 |
| 228 | 0.228 | $6.548 \mathrm{E}-01$ | $6.548 \mathrm{E}-02$ | $1.566 \mathrm{E}-02$ | 15.65590 | 3.56955 | 2.7508 |
| 230 | 0.23 | $9.914 \mathrm{E}-01$ | $9.914 \mathrm{E}-02$ | $1.643 \mathrm{E}-02$ | 16.42785 | 3.77841 | 2.7990 |
| 232 | 0.232 | $1.312 \mathrm{E}+00$ | 1.312E-01 | $1.538 \mathrm{E}-02$ | 15.38010 | 3.56818 | 2.7331 |
| 234 | 0.234 | $1.607 \mathrm{E}+00$ | $1.607 \mathrm{E}-01$ | $1.345 \mathrm{E}-02$ | 13.44752 | 3.14672 | 2.5988 |
| 236 | 0.236 | $1.850 \mathrm{E}+00$ | 1.850E-01 | $1.134 \mathrm{E}-02$ | 11.34020 | 2.67629 | 2.4284 |
| 238 | 0.238 | $2.060 \mathrm{E}+00$ | $2.060 \mathrm{E}-01$ | $9.909 \mathrm{E}-03$ | 9.90938 | 2.35843 | 2.2935 |
| 240 | 0.24 | $2.246 \mathrm{E}+00$ | $2.246 \mathrm{E}-01$ | $8.823 \mathrm{E}-03$ | 8.82328 | 2.11759 | 2.1774 |
| 242 | 0.242 | $2.413 \mathrm{E}+00$ | $2.413 \mathrm{E}-01$ | $7.991 \mathrm{E}-03$ | 7.99125 | 1.93388 | 2.0783 |
| 244 | 0.244 | $2.566 \mathrm{E}+00$ | $2.566 \mathrm{E}-01$ | $7.376 \mathrm{E}-03$ | 7.37608 | 1.79976 | 1.9982 |
| 246 | 0.246 | $2.708 \mathrm{E}+00$ | $2.708 \mathrm{E}-01$ | $6.891 \mathrm{E}-03$ | 6.89125 | 1.69525 | 1.9303 |
| 248 | 0.248 | $2.842 \mathrm{E}+00$ | 2.842E-01 | $6.450 \mathrm{E}-03$ | 6.45040 | 1.59970 | 1.8641 |
| 250 | 0.25 | $2.966 \mathrm{E}+00$ | $2.966 \mathrm{E}-01$ | $6.055 \mathrm{E}-03$ | 6.05493 | 1.51373 | 1.8009 |
| 252 | 0.252 | $3.084 \mathrm{E}+00$ | $3.084 \mathrm{E}-01$ | $5.739 \mathrm{E}-03$ | 5.73918 | 1.44627 | 1.7473 |
| 254 | 0.254 | $3.196 \mathrm{E}+00$ | $3.196 \mathrm{E}-01$ | $5.483 \mathrm{E}-03$ | 5.48285 | 1.39264 | 1.7016 |
| 256 | 0.256 | $3.303 \mathrm{E}+00$ | $3.303 \mathrm{E}-01$ | $5.240 \mathrm{E}-03$ | 5.24008 | 1.34146 | 1.6563 |
| 258 | 0.258 | $3.405 \mathrm{E}+00$ | $3.405 \mathrm{E}-01$ | $5.004 \mathrm{E}-03$ | 5.00420 | 1.29108 | 1.6103 |
| 260 | 0.26 | $3.503 \mathrm{E}+00$ | 3.503E-01 | $4.799 \mathrm{E}-03$ | 4.79877 | 1.24768 | 1.5684 |
| 262 | 0.262 | $3.597 \mathrm{E}+00$ | 3.597E-01 | $4.621 \mathrm{E}-03$ | 4.62140 | 1.21081 | 1.5307 |
| 264 | 0.264 | $3.688 \mathrm{E}+00$ | $3.688 \mathrm{E}-01$ | $4.450 \mathrm{E}-03$ | 4.45025 | 1.17487 | 1.4930 |
| 266 | 0.266 | $3.775 \mathrm{E}+00$ | $3.775 \mathrm{E}-01$ | $4.283 \mathrm{E}-03$ | 4.28338 | 1.13938 | 1.4547 |
| 268 | 0.268 | $3.859 \mathrm{E}+00$ | $3.859 \mathrm{E}-01$ | $4.131 \mathrm{E}-03$ | 4.13083 | 1.10706 | 1.4185 |
| 270 | 0.27 | $3.941 \mathrm{E}+00$ | $3.941 \mathrm{E}-01$ | $3.996 \mathrm{E}-03$ | 3.99645 | 1.07904 | 1.3854 |
| 272 | 0.272 | $4.019 \mathrm{E}+00$ | 4.019E-01 | $3.871 \mathrm{E}-03$ | 3.87098 | 1.05291 | 1.3535 |
| 274 | 0.274 | $4.095 \mathrm{E}+00$ | $4.095 \mathrm{E}-01$ | $3.747 \mathrm{E}-03$ | 3.74725 | 1.02675 | 1.3210 |
| 276 | 0.276 | $4.169 \mathrm{E}+00$ | $4.169 \mathrm{E}-01$ | $3.633 \mathrm{E}-03$ | 3.63333 | 1.00280 | 1.2901 |
| 278 | 0.278 | $4.241 \mathrm{E}+00$ | $4.241 \mathrm{E}-01$ | $3.532 \mathrm{E}-03$ | 3.53185 | 0.98185 | 1.2618 |
| 280 | 0.28 | $4.310 \mathrm{E}+00$ | $4.310 \mathrm{E}-01$ | $3.434 \mathrm{E}-03$ | 3.43363 | 0.96141 | 1.2336 |
| 282 | 0.282 | $4.378 \mathrm{E}+00$ | $4.378 \mathrm{E}-01$ | $3.330 \mathrm{E}-03$ | 3.33040 | 0.93917 | 1.2031 |


| 284 | 0.284 | $4.444 \mathrm{E}+00$ | 4.444E-01 | $3.228 \mathrm{E}-03$ | 3.22755 | 0.91662 | 1.1717 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 286 | 0.286 | $4.507 \mathrm{E}+00$ | 4.507E-01 | $3.133 \mathrm{E}-03$ | 3.13268 | 0.89595 | 1.1419 |
| 288 | 0.288 | $4.569 \mathrm{E}+00$ | $4.569 \mathrm{E}-01$ | $3.048 \mathrm{E}-03$ | 3.04792 | 0.87780 | 1.1145 |
| 290 | 0.29 | $4.629 \mathrm{E}+00$ | $4.629 \mathrm{E}-01$ | $2.971 \mathrm{E}-03$ | 2.97147 | 0.86173 | 1.0891 |
| 292 | 0.292 | $4.688 \mathrm{E}+00$ | $4.688 \mathrm{E}-01$ | $2.900 \mathrm{E}-03$ | 2.90005 | 0.84681 | 1.0647 |
| 294 | 0.294 | $4.745 \mathrm{E}+00$ | $4.745 \mathrm{E}-01$ | $2.831 \mathrm{E}-03$ | 2.83093 | 0.83229 | 1.0406 |
| 296 | 0.296 | $4.801 \mathrm{E}+00$ | 4.801E-01 | $2.763 \mathrm{E}-03$ | 2.76260 | 0.81773 | 1.0162 |
| 298 | 0.298 | $4.856 \mathrm{E}+00$ | $4.856 \mathrm{E}-01$ | $2.695 \mathrm{E}-03$ | 2.69540 | 0.80323 | 0.9915 |
| 300 | 0.3 | $4.909 \mathrm{E}+00$ | $4.909 \mathrm{E}-01$ | $2.631 \mathrm{E}-03$ | 2.63083 | 0.78925 | 0.9673 |
| 302 | 0.302 | $4.961 \mathrm{E}+00$ | $4.961 \mathrm{E}-01$ | $2.570 \mathrm{E}-03$ | 2.57037 | 0.77625 | 0.9441 |
| 304 | 0.304 | $5.012 \mathrm{E}+00$ | 5.012E-01 | $2.515 \mathrm{E}-03$ | 2.51450 | 0.76441 | 0.9221 |
| 306 | 0.306 | $5.061 \mathrm{E}+00$ | $5.061 \mathrm{E}-01$ | $2.462 \mathrm{E}-03$ | 2.46188 | 0.75333 | 0.9009 |
| 308 | 0.308 | $5.110 \mathrm{E}+00$ | $5.110 \mathrm{E}-01$ | $2.410 \mathrm{E}-03$ | 2.41037 | 0.74240 | 0.8798 |
| 310 | 0.31 | $5.158 \mathrm{E}+00$ | $5.158 \mathrm{E}-01$ | $2.359 \mathrm{E}-03$ | 2.35890 | 0.73126 | 0.8582 |
| 312 | 0.312 | $5.205 \mathrm{E}+00$ | 5.205E-01 | $2.309 \mathrm{E}-03$ | 2.30850 | 0.72025 | 0.8366 |
| 314 | 0.314 | $5.250 \mathrm{E}+00$ | 5.250E-01 | $2.261 \mathrm{E}-03$ | 2.26130 | 0.71005 | 0.8159 |
| 316 | 0.316 | $5.295 \mathrm{E}+00$ | 5.295E-01 | $2.218 \mathrm{E}-03$ | 2.21815 | 0.70094 | 0.7967 |
| 318 | 0.318 | $5.339 \mathrm{E}+00$ | 5.339E-01 | $2.177 \mathrm{E}-03$ | 2.17658 | 0.69215 | 0.7778 |
| 320 | 0.32 | $5.382 \mathrm{E}+00$ | 5.382E-01 | $2.133 \mathrm{E}-03$ | 2.13280 | 0.68250 | 0.7574 |
| 322 | 0.322 | $5.424 \mathrm{E}+00$ | $5.424 \mathrm{E}-01$ | $2.087 \mathrm{E}-03$ | 2.08695 | 0.67200 | 0.7357 |
| 324 | 0.324 | $5.465 \mathrm{E}+00$ | $5.465 \mathrm{E}-01$ | $2.044 \mathrm{E}-03$ | 2.04403 | 0.66226 | 0.7149 |
| 326 | 0.326 | $5.506 \mathrm{E}+00$ | 5.506E-01 | $2.008 \mathrm{E}-03$ | 2.00773 | 0.65452 | 0.6970 |
| 328 | 0.328 | $5.546 \mathrm{E}+00$ | 5.546E-01 | $1.973 \mathrm{E}-03$ | 1.97333 | 0.64725 | 0.6797 |
| 330 | 0.33 | $5.585 \mathrm{E}+00$ | $5.585 \mathrm{E}-01$ | $1.931 \mathrm{E}-03$ | 1.93140 | 0.63736 | 0.6582 |
| 332 | 0.332 | $5.623 \mathrm{E}+00$ | 5.623E-01 | $1.882 \mathrm{E}-03$ | 1.88242 | 0.62497 | 0.6326 |
| 334 | 0.334 | $5.660 \mathrm{E}+00$ | $5.660 \mathrm{E}-01$ | $1.842 \mathrm{E}-03$ | 1.84200 | 0.61523 | 0.6109 |
| 336 | 0.336 | $5.697 \mathrm{E}+00$ | 5.697E-01 | $1.822 \mathrm{E}-03$ | 1.82203 | 0.61220 | 0.5999 |
| 338 | 0.338 | $5.733 \mathrm{E}+00$ | 5.733E-01 | $1.809 \mathrm{E}-03$ | 1.80862 | 0.61132 | 0.5926 |
| 340 | 0.34 | $5.769 \mathrm{E}+00$ | 5.769E-01 | $1.775 \mathrm{E}-03$ | 1.77522 | 0.60358 | 0.5739 |
| 342 | 0.342 | $5.804 \mathrm{E}+00$ | 5.804E-01 | $1.726 \mathrm{E}-03$ | 1.72580 | 0.59022 | 0.5457 |
| 344 | 0.344 | $5.838 \mathrm{E}+00$ | $5.838 \mathrm{E}-01$ | $1.702 \mathrm{E}-03$ | 1.70238 | 0.58562 | 0.5320 |
| 346 | 0.346 | $5.872 \mathrm{E}+00$ | 5.872E-01 | $1.699 \mathrm{E}-03$ | 1.69860 | 0.58772 | 0.5298 |
| 348 | 0.348 | $5.906 \mathrm{E}+00$ | 5.906E-01 | $1.647 \mathrm{E}-03$ | 1.64728 | 0.57325 | 0.4991 |
| 350 | 0.35 | $5.938 \mathrm{E}+00$ | $5.938 \mathrm{E}-01$ | $1.579 \mathrm{E}-03$ | 1.57925 | 0.55274 | 0.4570 |
| 352 | 0.352 | $5.969 \mathrm{E}+00$ | 5.969E-01 | $1.588 \mathrm{E}-03$ | 1.58815 | 0.55903 | 0.4626 |
| 354 | 0.354 | $6.002 \mathrm{E}+00$ | 6.002E-01 | $1.628 \mathrm{E}-03$ | 1.62758 | 0.57616 | 0.4871 |
| 356 | 0.356 | $6.034 \mathrm{E}+00$ | $6.034 \mathrm{E}-01$ | $1.587 \mathrm{E}-03$ | 1.58707 | 0.56500 | 0.4619 |
| 358 | 0.358 | $6.065 \mathrm{E}+00$ | $6.065 \mathrm{E}-01$ | $1.453 \mathrm{E}-03$ | 1.45287 | 0.52013 | 0.3735 |
| 360 | 0.36 | $6.092 \mathrm{E}+00$ | 6.092E-01 | $1.368 \mathrm{E}-03$ | 1.36808 | 0.49251 | 0.3134 |
| 362 | 0.362 | $6.120 \mathrm{E}+00$ | 6.120E-01 | $1.440 \mathrm{E}-03$ | 1.44010 | 0.52132 | 0.3647 |
| 364 | 0.364 | $6.150 \mathrm{E}+00$ | $6.150 \mathrm{E}-01$ | $1.549 \mathrm{E}-03$ | 1.54875 | 0.56374 | 0.4374 |
| 366 | 0.366 | $6.182 \mathrm{E}+00$ | 6.182E-01 | $1.525 \mathrm{E}-03$ | 1.52530 | 0.55826 | 0.4222 |
| 368 | 0.368 | $6.211 \mathrm{E}+00$ | 6.211E-01 | $1.351 \mathrm{E}-03$ | 1.35118 | 0.49723 | 0.3010 |
| 370 | 0.37 | $6.236 \mathrm{E}+00$ | $6.236 \mathrm{E}-01$ | $1.258 \mathrm{E}-03$ | 1.25775 | 0.46537 | 0.2293 |
| 372 | 0.372 | $6.261 \mathrm{E}+00$ | $6.261 \mathrm{E}-01$ | $1.361 \mathrm{E}-03$ | 1.36143 | 0.50645 | 0.3085 |
| 374 | 0.374 | $6.290 \mathrm{E}+00$ | $6.290 \mathrm{E}-01$ | $1.447 \mathrm{E}-03$ | 1.44725 | 0.54127 | 0.3697 |
| 376 | 0.376 | $6.319 \mathrm{E}+00$ | $6.319 \mathrm{E}-01$ | $1.355 \mathrm{E}-03$ | 1.35513 | 0.50953 | 0.3039 |
| 378 | 0.378 | $6.344 \mathrm{E}+00$ | $6.344 \mathrm{E}-01$ | $1.236 \mathrm{E}-03$ | 1.23558 | 0.46705 | 0.2115 |
| 380 | 0.38 | $6.369 \mathrm{E}+00$ | $6.369 \mathrm{E}-01$ | $1.258 \mathrm{E}-03$ | 1.25765 | 0.47791 | 0.2292 |
| 382 | 0.382 | $6.395 \mathrm{E}+00$ | $6.395 \mathrm{E}-01$ | $1.326 \mathrm{E}-03$ | 1.32632 | 0.50666 | 0.2824 |
| 384 | 0.384 | $6.422 \mathrm{E}+00$ | 6.422E-01 | $1.299 \mathrm{E}-03$ | 1.29947 | 0.49900 | 0.2620 |
| 386 | 0.386 | $6.447 \mathrm{E}+00$ | $6.447 \mathrm{E}-01$ | $1.216 \mathrm{E}-03$ | 1.21573 | 0.46927 | 0.1953 |


| 388 | 0.388 | $6.470 \mathrm{E}+00$ | $6.470 \mathrm{E}-01$ | $1.196 \mathrm{E}-03$ | 1.19592 | 0.46402 | 0.1789 |
| :--- | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 390 | 0.39 | $6.495 \mathrm{E}+00$ | $6.495 \mathrm{E}-01$ | $1.245 \mathrm{E}-03$ | 1.24530 | 0.48567 | 0.2194 |
| 392 | 0.392 | $6.520 \mathrm{E}+00$ | $6.520 \mathrm{E}-01$ | $1.246 \mathrm{E}-03$ | 1.24578 | 0.48834 | 0.2198 |
| 394 | 0.394 | $6.544 \mathrm{E}+00$ | $6.544 \mathrm{E}-01$ | $1.155 \mathrm{E}-03$ | 1.15475 | 0.45497 | 0.1439 |
| 396 | 0.396 | $6.566 \mathrm{E}+00$ | $6.566 \mathrm{E}-01$ | $1.114 \mathrm{E}-03$ | 1.11430 | 0.44126 | 0.1082 |
| 398 | 0.398 | $6.589 \mathrm{E}+00$ | $6.589 \mathrm{E}-01$ | $1.187 \mathrm{E}-03$ | 1.18715 | 0.47249 | 0.1716 |
| 400 | 0.4 | $6.614 \mathrm{E}+00$ | $6.614 \mathrm{E}-01$ |  |  |  |  |

Integration Using trapazoidal rule from 0.208 to 0.398

|  | $\mathrm{E}(\Theta)$ | $\Theta^{*} \mathrm{E}(\Theta)$ |
| :---: | :---: | :---: |
| Value of $n$ | 95 | 95 |
| b-a | 0.1900 | 0.1900 |
| f(.208) | 0.0017 | 0.0007 |
| f(.398) | 1.1871 | 0.4725 |
| Sum f(.21)-f(.396) | 328.8818 | 89.6471 |
| Integral | 0.6590 | 0.1798 |
| Tail | 0.3410 | 0.2337 |
| Total | 1.0000 | 0.4135 |

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[^0]:    ${ }^{1}$ Chemical Engineering Education, 38, No 2 Spring 2004 pp 154-160

